

(12) INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(19) World Intellectual Property Organization
International Bureau



(43) International Publication Date
14 November 2002 (14.11.2002)

PCT

(10) International Publication Number
WO 02/091554 A1

(51) International Patent Classification⁷: H02K 49/04, E21B 7/06, H02K 16/02, E21B 17/02, H02K 7/18, F04D 29/66, H02K 17/34, F04D 13/10, E21B 41/00

(21) International Application Number: PCT/GB02/01993

(22) International Filing Date: 1 May 2002 (01.05.2002)

(25) Filing Language: English

(26) Publication Language: English

(30) Priority Data:
0111124.4 5 May 2001 (05.05.2001) GB

(71) Applicant and

(72) Inventor: SPRING, Gregson, William, Martin [GB/GB]; 7 Great Calcroft, Defford Road, Pershore, Worcestershire WR10 1QS (GB).

(74) Agent: JACKSON, Derek, Charles; Derek Jackson Associates, The Old Yard, Lower Town, Claines, Worcester WR3 7RY (GB).

(81) Designated States (national): AE, AG, AL, AM, AT, AU, AZ, BA, BB, BG, BR, BY, BZ, CA, CH, CN, CO, CR, CU, CZ, DE, DK, DM, DZ, EC, EE, ES, FI, GB, GD, GE, GH, GM, HR, HU, ID, IL, IN, IS, JP, KE, KG, KP, KR, KZ, LC, LK, LR, LS, LT, LU, LV, MA, MD, MG, MK, MN, MW, MX, MZ, NO, NZ, OM, PH, PL, PT, RO, RU, SD, SE, SG, SI, SK, SL, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VN, YU, ZA, ZM, ZW.

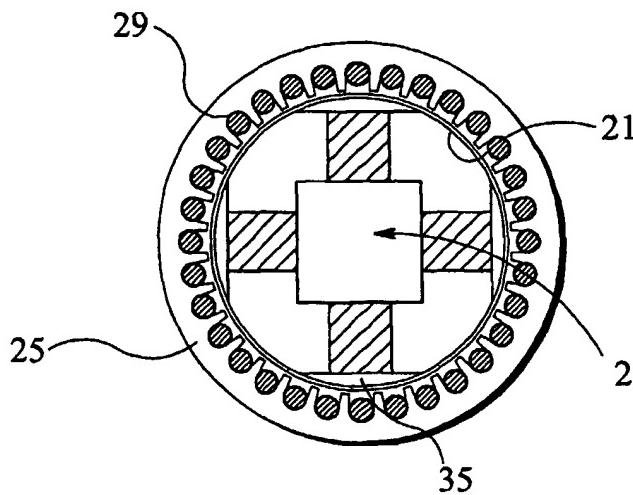
(84) Designated States (regional): ARIPO patent (GH, GM, KE, LS, MW, MZ, SD, SL, SZ, TZ, UG, ZM, ZW), Eurasian patent (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European patent (AT, BE, CH, CY, DE, DK, ES, FI, FR, GB, GR, IE, IT, LU, MC, NL, PT, SE, TR), OAPI patent (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, ML, MR, NE, SN, TD, TG).

Published:

— with international search report

For two-letter codes and other abbreviations, refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.

(54) Title: DOWNHOLE TORQUE GENERATOR



(57) Abstract: A torque-generating apparatus comprises a first assembly (10, 25) including a generally cylindrical member of magnetically soft material and having a longitudinal axis and a second assembly (2) arranged coaxially within the first assembly and including an electromagnetic winding (4). The first assembly (10, 25) and the second assembly (2) are rotatable relative to each other about the axis. Relative rotation between the first (10, 25) and second (2) assemblies induces a magnetic field which generates rotational torque between the first and second assemblies.

WO 02/091554 A1

BEST AVAILABLE COPY

- 1 -

DOWNHOLE TORQUE GENERATOR

This invention relates to the field of torque-generating apparatus. More particularly, but not exclusively, the
5 invention relates such apparatus for down hole use in drilling applications.

In oilfield exploration or extraction applications, a drill string is used to drill a well bore. In order to provide
10 directional control of the drill string, a bent section of drill collar (commonly referred to as a bent sub) may be used. This section of the drill string is called the orienter. The bent sub may be rotated about the longitudinal axis of the drill string so as to provide directional
15 control. The orientation of the bent sub is commonly controlled by a conventional electric motor and reduction gear box, powered from the surface via a wire line connection. Directional control of the drill string is achieved by rotating the bent sub, straight drilling being
20 achieved by continuously rotating the bent sub so as to eliminate any drilling bias.

The present system has many disadvantages. The power for the down hole electric motor is commonly provided by a very high-voltage, high-current power supply (e.g. 1KV with 3 Amps current capacity). Voltages and currents of this magnitude are needed to overcome the high electrical losses in the very long cables used to supply the down hole instrumentation.
The arrangement is not power-efficient and requires
25 specialist equipment in order for the wire line apparatus to function. Furthermore, although this system has been used fairly satisfactorily for many years, its performance and reliability is compromised at high ambient working temperatures which may exceed 150 C. Conduction losses in
30 the copper windings and magnetic hysteresis losses in the

- 2 -

ferrous frame of the motor add to the thermal stress imposed by this high ambient temperature. These and other factors lead to a shortening of the working life of an electrical motor in these conditions. This is a major drawback in
5 drilling applications - should the motor fail down hole, the string cannot be steered, and drilling may be prematurely stopped. The motor then has to be replaced, which can lead to significant and costly drilling downtime. A more reliable motor would therefore lead to decreased downtime, giving
10 lower operating costs.

One conventional approach to down hole drilling uses a rotating drill string. In such an apparatus, a steering tool is required which is able to maintain a stable position in
15 relation to a rotating drilling pipe. One proposed solution has been to provide two electrical generators, mounted to the top and bottom of the steering tool respectively, the generators having opposing turbine blades and rotating in opposite directions in the flow of drilling mud so as to
20 cancel some of the natural background torque introduced by the turbines. This tactic allows the tool some measure of platform stability.

However, in reality one generator always exhibits more
25 rotational loss than the other, resulting in a net torque which slowly rotates the steering tool. To eliminate this error, an onboard microprocessor is provided so that an electrical load might be applied to one or other generator so as to correct the undesired rotation. This solution uses
30 conventional electrical generators and, although it has been very effective up to a point, presents some practical problems.

In the conventional apparatus, the stator is an integral part
35 of the steering tool and the rotor is allowed freely to

- 3 -

- rotate in the drilling mud. To achieve the required levels of torque, short circuit armature current is drawn from the electrical generators. This causes a considerable heating effect within the armature winding space. Whilst the
- 5 drilling mud removes some heat, the winding temperature cannot be kept within safe limits at the high ambient temperatures encountered in down hole drilling. In ambient temperatures of the region of 150 C, the winding insulation suffers severe thermal stress and usually fails after only a
- 10 few hours, leading to costly downtime. This failure is symptomatic of the design of the apparatus and cannot be avoided. This type of tool is therefore not suitable for high-premium, high-temperature wells.
- 15 Furthermore, present apparatus is vulnerable to the aggregation of mud and debris leading to clogging of the machinery and down hole failure.
- There is therefore a requirement for an improved apparatus
- 20 avoiding the above disadvantages. In particular, there is a requirement for an apparatus that is reliable and efficient in operation, with an improved lifetime. There is in addition a requirement for an apparatus which is less susceptible to thermal failure and clogging problems.
- 25 It is an object of the present invention to provide a torque-generating apparatus capable of fulfilling at least some of the above requirements.
- 30 According to the present invention there is provided a torque-generating apparatus comprising: a first assembly including a generally cylindrical member of magnetically soft material and having a longitudinal axis; a second assembly arranged coaxially within the first assembly and including an electromagnetic winding, the first assembly and the second
- 35

- 4 -

assembly being rotatable relative to each other about the axis; the arrangement being such that relative rotation between the first and second assemblies induces a magnetic field which generates rotational torque between the first and
5 second assemblies.

A "magnetically soft material" is a material which is not capable of being substantially permanently magnetised, but which becomes magnetised whilst in an externally applied
10 magnetic field.

The first assembly may be a rotor assembly of the apparatus for producing rotational torque and the second assembly may be a stator assembly of the apparatus.

15 The second assembly may comprise a magnetically soft steel.

The first and second assemblies may be separated by a narrow gap.

20 The first assembly may be disposed so as substantially to surround the second assembly.

25 The first assembly may be substantially solid or may be formed from a plurality of laminations.

The second assembly may be substantially solid or may be formed from a plurality of laminations.

30 A number of generally longitudinal grooves may be provided to the inside surface of the cylindrical body of the first assembly. The grooves may be substantially coaxial with the longitudinal axis of the first assembly. Alternatively, the grooves may be provided so as to form at least a partial
35 helix around the longitudinal axis of the first assembly.

- 5 -

The second assembly may be provided with a number of pole pieces extending generally radially from the longitudinal axis thereof. The pole pieces of the second assembly may be provided with an electromagnetic winding, adjacent poles

- 5 being magnetisable in opposite directions. Means may be provided to control the degree of the magnetisation. Gaps between the pole pieces may be filled with a potting material. The surface of the second assembly may be covered with a layer of soft magnetic or non-magnetic material.

10

The second assembly may be provided with an external power supply. The power supply may comprise a direct current power supply in the form of a battery, or a generator provided with rectification means.

15

The generator may be provided with external rotation means adapted to rotate contrary to the rotational direction of the first assembly.

- 20 The first assembly may be provided with external rotation means such as impeller means adapted to rotate the first assembly, the impeller means being adapted to be disposed in use within a moving fluid, the motion of the fluid acting upon the impeller means so as to rotate the first assembly.

- 25 The impeller means may be demountably attached to the first assembly.

- 30 The invention may comprise a torque-generating apparatus as described above for use in a down hole drilling apparatus, the fluid which acts to rotate the impeller being drilling mud. The down hole drilling apparatus may include a drill head, the drill head being provided with associated stabilising means which include a torque-generating apparatus as described above, the torque-generating apparatus acting to provide a controllable amount of torque to stabilise the

- 6 -

orientation of the drill head. Wire line drilling apparatus may be provided including a rotatable bent sub, rotation of the bent sub being effected using a torque-generating apparatus according to the invention.

5

For a better understanding of the present invention and to show more clearly how it may be carried into effect, reference will now be made, by way of example, to the accompanying drawings which show schematically various 10 embodiments of the present invention. The figures may not be to scale. In the drawings:

Figure 1 is an elevational view, partly in section, of a torque-generating apparatus according to the present 15 invention;

Figure 2 is cross-sectional view of the apparatus of Figure 1;

20 Figure 3 is an end view of the apparatus of Figure 1 looking in the direction of the arrow B in Figure 1;

Figure 4 is a cross-sectional view of the apparatus of Figure 1 taken along the line A-A shown in Figure 2;

25 Figure 5 is a more detailed view of a stator shown in Figure 2;

Figure 6 shows two views of a stator pole piece;

30 Figure 7 is a cross-section of the stator shown in Figure 5;

Figure 8 is an end view of the stator shown in Figure 5;

35 Figure 9 is a sectional view of a rotor shown in Figure 2;

- 7 -

Figure 10 is an end view of the rotor of Figure 9;

Figure 11 is an end view detail of the rotor and stator assembly; and

5

Figure 12 is a cross-sectional view of another embodiment of a rotor similar to that shown in Figure 2 showing a laminated structure;

10 Figure 13 is an end view of the rotor of Figure 12; and

Figure 14 is an end view detail of a rotor and stator assembly incorporating the rotor of Figures 12 and 13.

15 Figures 1 and 2 show a torque generating apparatus suitable for use in drilling apparatus. As shown in Figure 2, the apparatus comprises a second assembly in the form of a cylindrical stator 2 which is rotatable about its longitudinal axis. The stator 2 is shown in more detail in Figures 5 to 8. The stator is made of magnetically soft material and is provided with stator windings 4 arranged so that, when energised, the stator 2 is magnetised as discussed below. The stator is supported on bearings 6, such as thrust bearings, and has an output shaft 8. Surrounding the stator 20 is a first assembly in the form of a magnetically soft steel rotor 10. The rotor 10 is mounted on the stator 2 by way of bearings 11, such as thrust bearings. The rotor has associated with it impeller blades 12, rotation of which cause the rotor 10 to rotate. The rotor/stator assembly is 25 contained within a cylindrical housing 14 which may typically be a section of drill collar. The assembly is supported by an anchor and bearings 16 which are bolted to the housing. An electrical supply, in this case an electrical alternator/generator 18, is provided so that the stator 30 windings 4 may be energised. The electrical supply may 35

- 8 -

- alternatively be a battery or other direct current power source (none shown). The power source may rotate with the apparatus, or may be stationary, being connected to the stator windings 4 via e.g. slip rings (not shown). The
- 5 stator windings 4 are connected to windings of the alternator by way of access holes 19 formed in the core of each apparatus as shown in Figure 2. In use, the apparatus is disposed down hole, drilling mud being pumped down the hole in the direction indicated. The moving drilling fluid acts
- 10 on the impeller 12 so as to rotate the rotor 10. Even with no electrical current supplied, some rotation of the stator 2 occurs, due to residual magnetism in the rotor 10 and stator bodies and residual coupling torque between the rotor and stator bearings. However, upon energisation of the stator
- 15 windings 4, stator rotation is increased, the amount of rotation being controllable by controlling the current supplied to the stator windings. This control may be suitably effected by onboard circuitry (not shown) controlled by e.g. a microprocessor, and may conveniently be exercised
- 20 remotely, e.g. from the surface. Alternatively or additionally, a closed-loop control system may be provided so that the electromagnetic field strength may be adjusted to maintain a constant torque, avoiding torque ripple due to small variations in mud-pump and rotor speeds.
- 25
- In practice, operating efficiency is maximised by employing moderate electromagnetic drive/voltage current together with the highest practical rotor speed for all mud flow rates. The magnitude of output torque is a function of machine size
- 30 and may be scaled for bigger or smaller machines.

Figure 3 is an end view of the apparatus of Figure 1. It shows the output shaft 8 with associated bearing 6 and also shows the impeller blades 12. The direction of mud flow in

- 9 -

Figure 3 is into the paper, causing rotation of the impeller blades.

Figure 4 shows a cross-sectional view as indicated in Figure 5 2. The assembly anchor and bearing 16 is shown in cross-section, bolted to the drill collar housing. The output shaft 8 can be seen in cross-section. The view in Figure 4 is looking inwards into the assembly in the direction of the incoming mud, and the impeller blades 12 can be seen behind 10 the assembly anchor 16.

The impeller 12 is a component which suffers high wear and needs frequent replacement. As such it is demountably attached to the rotor 10, in this case having a screw thread 15 which co-operates with a corresponding thread on the rotor.

Figures 5 to 8 are more detailed views of the stator assembly. The stator 2 is a simple four-pole electromagnet which forms the electrical and mechanical centre of the 20 machine. The stator 2 has a central shaft from which radially project four pole pieces 35, as shown. The number of pole pieces need not be limited to four - any suitable number of pole pieces may be provided, larger machines requiring more pole pieces.

25 To prevent the stator 2 from being crushed by normal down hole drilling pressures, the gaps between the stator pole pieces may be filled with a high compressive strength material such as epoxy filler (not shown). This allows the 30 stator 2 to maintain its shape and survive pressures in excess of 20,000 pounds per square inch.

The stator 2 is wound with high-temperature-resistant 35 enamelled copper wire (not shown in Figures 5 to 8) so as to produce alternate north and south magnetisation of the pole

- 10 -

pieces. To preserve the integrity of the stator winding from the drilling mud, as shown in Figures 2 and 11 a thin sleeve of soft magnetic or non-magnetic material 21 is machined to cover the stator windings. End cheeks provided on the stator 5 receive the sleeve 21 and are welded thereto to seal the assembly. This seals the edges of the stator 2 and protects the contents from contamination. The covering, for example in the form of a cylinder, allows the stator poles to rotate with respect to the rotor 10 whilst maintaining close 10 magnetic contact. A small magnetic gap is required to create the high output torque reactions from this machine.

An important feature of the apparatus is the use of electromagnetic advantage to minimise the power demands of 15 the stator 2. By their nature, induction machines rely upon high rates of change of magnetic flux to rotate the rotor or stator. High operating efficiency will therefore be achieved at high RPM.

20 Figures 9 to 11 show the rotor 10 in more detail. The rotor 10 consists of a simple steel cylinder having grooves 20 machined to the inside surface. The grooves 20 perform two important functions. They allow the rotor 10 and stator 2 to maintain close magnetic contact and at the same time allow a 25 sufficient flow of drilling mud through the annular space between the rotor and stator. This aids lubrication of the rotor bearings 11 and allows dissipation of heat.

The grooves 20 also prevent mud particles from aggregating 30 within the annular space and clogging the apparatus. If the annular space were too small, mud particles would become trapped due to low mud-flow velocities. The mud particles would quickly aggregate, binding the stator 2 and rotor 10 and causing a down hole failure. In conventional electrical 35 apparatus using a permanent magnet rotor, failure frequently

- 11 -

- occurs due to mud material becoming trapped and clogged within the space between the rotor and stator. The clogging problem is compounded by both soft and hard magnetic particles that circulate within the mud. Once trapped by the
- 5 strong magnetic fields within the permanent magnet rotor, the magnetic particles capture non-magnetic mud particles, accelerating clogging. A conventional permanent magnet torque-generating apparatus (torquer/motor) can seize and fail very quickly. The present apparatus avoids this type of
- 10 failure by providing a more generous space between rotor and stator (due to the grooves) and by being composed of soft magnetic material which does not trap particles to the same extent as a permanent magnet.
- 15 An important feature of the present apparatus is the use of electromagnetic advantage and a solid rotor 10 to dissipate waste heat from work done by the apparatus. The induced currents circulating in the rotor 10 would give rise to I^2R heating in the rotor raising its working temperature.
- 20 However, because the rotor 10 is manufactured from a magnetically soft material, its performance is unaffected by this temperature rise. It can therefore operate in temperatures much higher than the current limit of 180 C, without any loss of performance. In theory, the rotor 10
- 25 alone can operate at temperatures up to 500 C, but in practice this is limited by the temperature limit of the rotor support bearings 11.
- According to another realisation of the rotor 10, not shown
- 30 in the figures, the grooves 20 are formed with a small flute or spiral twist along their length. In this way, every rotation of the rotor 10 produces a small pumping effect, pumping mud and contaminant particles through the apparatus. Should the apparatus begin to foul with magnetic particles,
- 35 the blockage may be cleared by allowing the apparatus to run

- 12 -

without electromagnetic excitation. With no magnetic attraction, particles are easily expelled. Normal energisation of the electromagnet can then be resumed. This feature could be programmed into the apparatus as a general 5 housekeeping function, which would further improve the reliability of the apparatus.

The pole pieces of the stator 2 and the protruding portions of the rotor inner surface are disposed so as to correspond, 10 being aligned (in the case of a four-pole apparatus) every quarter turn of the rotor 10. As discussed, the number of pole pieces and protruding portions may be varied to suit a particular application. Although the rotor 10 and stator 2 in this example are formed of magnetically soft steel, any 15 suitable soft magnetic material may be employed. Similarly, the protective coating of the stator 2 may be made of ferrous or non-ferrous material.

Normal use of the apparatus may erode the inner surface of 20 the rotor 10 and/or the protective coating of the stator 2. This would cause a gradual loss of output torque. The apparatus is, however, easy and economical to repair, as any mechanical errors may be easily corrected by welding, machining and/or grinding the relevant part.

25 For the generation of output torque, the present invention requires an external power source to excite the electromagnet. Whilst batteries are commonly used to power down hole instruments, an alternative solution is to add a 30 small down hole alternator/generator to the apparatus. The generator rotor (see Figures 1 and 2) is allowed to rotate freely and independently of the rotor 10 and may incorporate a power-conditioning electronic circuit to regulate power to the stator electromagnet. Due to the high operating 35 efficiency of the apparatus, the generator may be of modest

- 13 -

design with an output of say, 50 Watts. Any suitable conventional generator may be employed, e.g. a multipole permanent-magnet rotor and a three-phase wound stator. Such generators are commonly provided down hole to power down hole
5 instrumentation. Although in Figures 1 and 2 the generator and torque-generating apparatus are shown in close physical proximity, they may in fact be located some distance apart.

In situations where a generator is not suitable, current may
10 be provided via a wire line. This would necessitate the provision of e.g. slip rings or bearings so as to maintain electrical contact between the apparatus and the non-rotating wire line connection.

15 As discussed, the apparatus may be used as a torquer, providing a controlled torque so as to stabilise the drilling apparatus. An apparatus according to the invention may alternatively be used as a motor, providing rotation of the bent sub. The only difference between the apparatus used as
20 a torquer or as a motor is the stator RPM. A potential problem of the apparatus used as a motor is the loss of differential speed between the generator rotor and stator. The torque reaction between any rotor and stator pair drags the stator around its axis in the same direction as the
25 rotor. If both the torquer and generator impeller blades rotate in the same direction, this leads to a reduced differential speed between the generator rotor and stator leading to loss of output. However, if the blades are arranged so as to rotate in opposite directions, this
30 prevents generator output failure when the apparatus is operated as a motor, as the differential speed is increased. The system is therefore self-compensating to some degree and its stability is improved. Use of the apparatus as a motor may necessitate the provision of a reduction gearbox for
35 increased torque.

- 14 -

Figures 12 to 14 show an alternative rotor 25 in more detail. The rotor 25 consists of a laminated steel cylinder having a number of conductors 29 running the length of the rotor 25. The conductors 29 are connected at each end of the rotor 25
5 by means of a conductor end cap 27. The arrangement of conductors 29 and end caps 27 form what is known as a squirrel cage conductor winding.

The conductors 29 and conductor end caps 27 consist of rods
10 and plates of beryllium copper, which has a similar electrical resistivity to aluminium but is a stronger material, resistant to mechanical abrasion and chemical attack from drilling muds.

15 A close magnetic contact is still maintained between the rotor 25 and the stator 2.

It should be understood that the stator which can be in a solid or laminated form can be used in conjunction with
20 either a solid or laminated rotor.

- 15 -

CLAIMS

1. A torque-generating apparatus comprising: a first assembly (10, 25) including a generally cylindrical member of magnetically soft material and having a longitudinal axis; a second assembly (2) arranged coaxially within the first assembly and including an electromagnetic winding (4), the first assembly (10, 25) and the second assembly (2) being rotatable relative to each other about the axis; the arrangement being such that relative rotation between the first (10, 25) and second (2) assemblies induces a magnetic field which generates rotational torque between the first and second assemblies.
- 15 2. An apparatus as claimed in claim 1, characterised in that the first assembly is a rotor assembly (10, 25) of the apparatus for producing rotational torque and the second assembly is a stator assembly (2) of the apparatus.
- 20 3. An apparatus as claimed in claim 1 or 2, characterised in that the second assembly (2) comprises a magnetically soft steel.
- 25 4. An apparatus as claimed in any preceding claim, characterised in that the first (10, 25) and second (2) assemblies are separated by a narrow gap.
- 30 5. An apparatus as claimed in any preceding claim, characterised in that the first assembly (10, 25) is disposed so as substantially to surround the second assembly (2).
6. A combination as claimed in any preceding claim, characterised in that the first assembly is substantially solid (10) or is formed from a plurality of laminations (25).

- 16 -

7. An apparatus as claimed in any preceding claim, characterised in that the second assembly (2) is substantially solid or is formed from a plurality of laminations.

5

8. An apparatus as claimed in any preceding claim, characterised in that a number of generally longitudinal grooves (20) are provided to the inside surface of the cylindrical body of the first assembly (10).

10

9. An apparatus as claimed in claim 8, characterised in that the grooves (20) are substantially coaxial with the longitudinal axis of the first assembly (10).

15

10. An apparatus as claimed in claim 8, characterised in that the grooves (20) are provided so as to form at least a partial helix around the longitudinal axis of the first assembly (10).

20

11. An apparatus as claimed in any preceding claim, characterised in that the second assembly (2) is provided with a number of pole pieces extending generally radially from the longitudinal axis thereof.

25

12. An apparatus as claimed in claim 11, characterised in that the pole pieces of the second assembly (2) are provided with an electromagnetic winding, adjacent poles being magnetisable in opposite directions.

30

13. An apparatus as claimed in claim 11 or 12, characterised in that means is provided to control the degree of the magnetisation.

- 17 -

14. An apparatus as claimed in any one of claims 11 to 13, characterised in that gaps between the pole pieces are filled with a potting material.
- 5 15. An apparatus as claimed in any one of claims 11 to 14, characterised in that the surface of the second assembly (2) is covered with a layer of soft magnetic or non-magnetic material.
- 10 16. An apparatus as claimed in any preceding claim, characterised in that the second assembly is provided with an external power supply (18).
- 15 17. An apparatus as claimed in claim 16, characterised in that the power supply (18) comprises a direct current power supply in the form of a battery, or a generator provided with rectification means.
- 20 18. An apparatus as claimed in claim 17, characterised in that the generator is provided with external rotation means adapted to rotate contrary to the rotational direction of the first assembly.
- 25 19. An apparatus as claimed in any preceding claim, characterised in that the first assembly (10, 25) is provided with external rotation means (12) adapted to rotate the first assembly.
- 30 20. An apparatus as claimed in claim 19, characterised in that the external rotation means comprises impeller means (12).
- 35 21. An apparatus as claimed in claim 20, characterised in that the impeller means (12) is adapted to be disposed in use within a moving fluid, the motion of the fluid acting upon

- 18 -

the impeller means (12) so as to rotate the first assembly (10, 25).

22. An apparatus as claimed in claim 20 or 21, characterised
5 in that the impeller means (12) is demountably attached to
the first assembly (10, 25).

23. An apparatus as claimed in any one of claims 20 to 22,
for use in a down hole drilling apparatus, the fluid which
10 acts to rotate the impeller (12) being drilling mud.

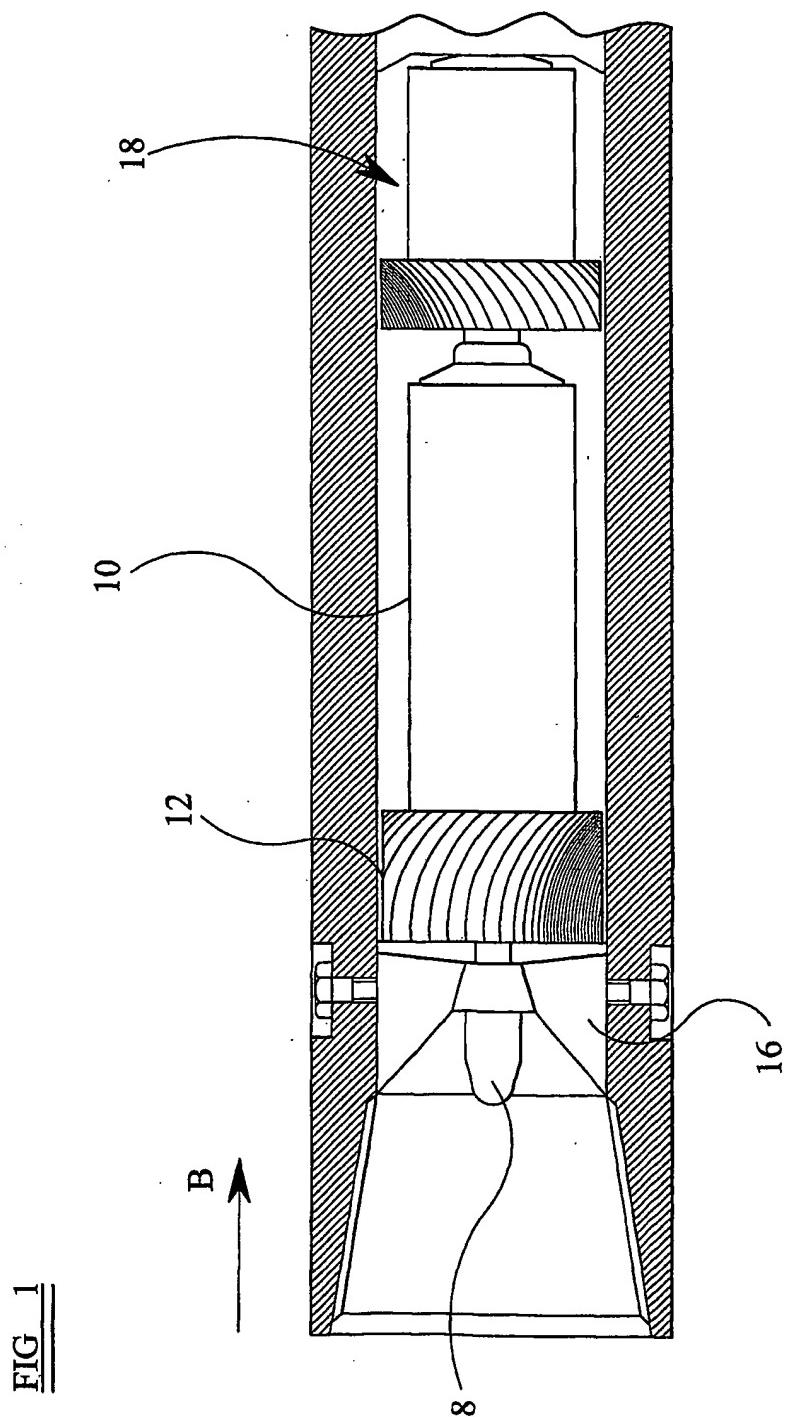


FIG. 1

2 / 6

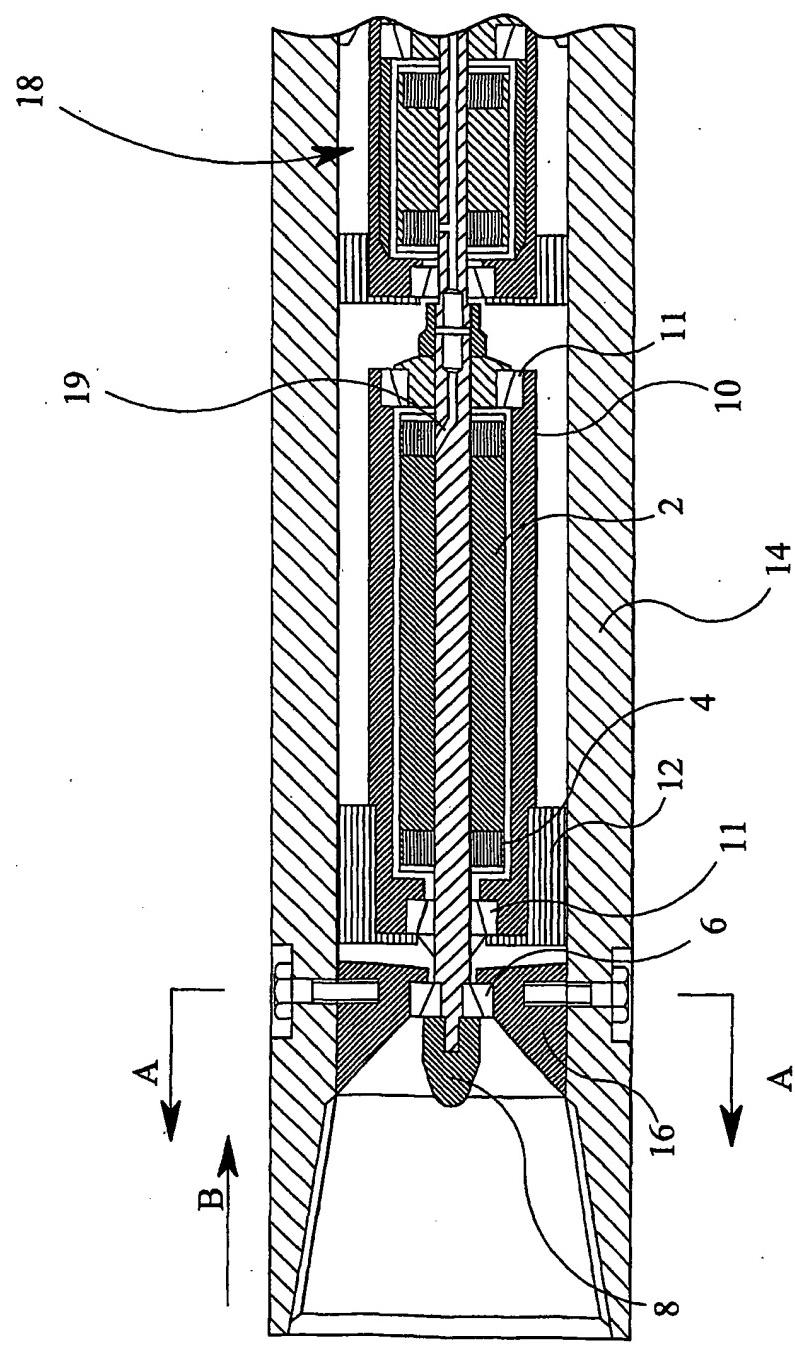
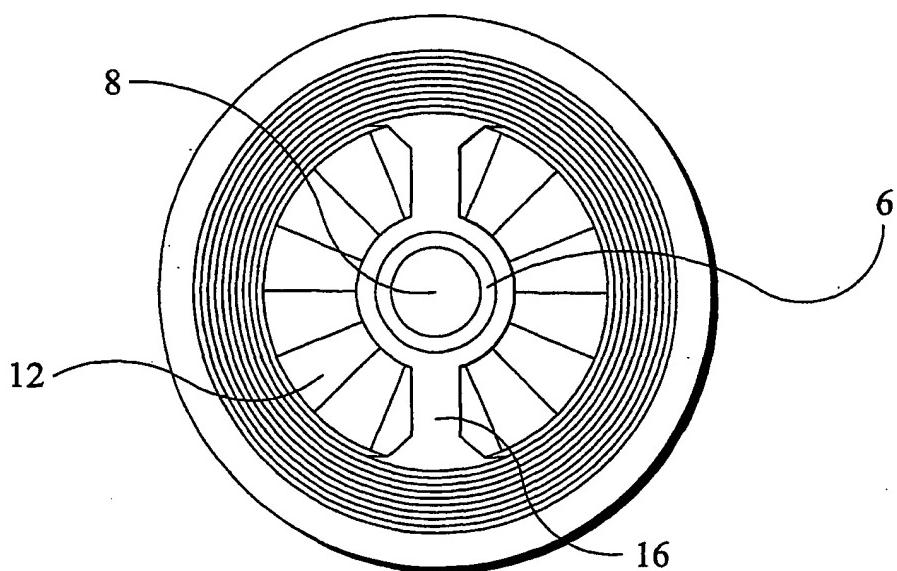
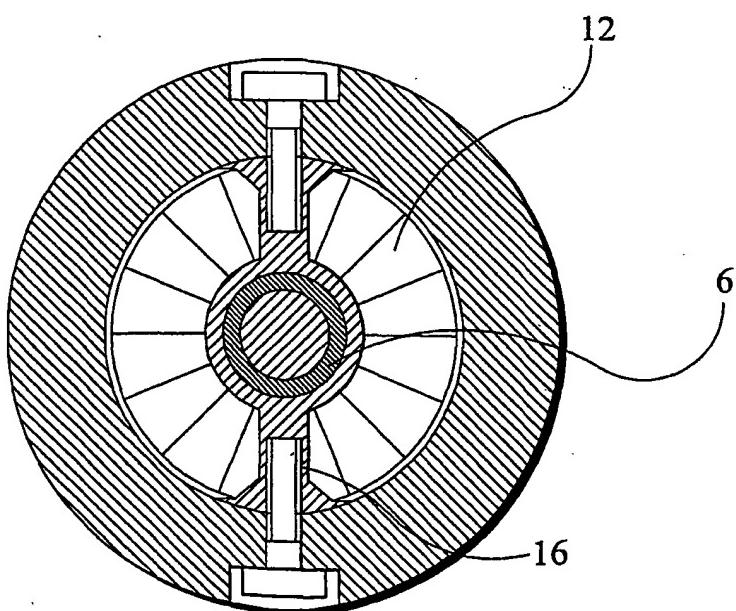


FIG 2

3 / 6

FIG 3FIG 4

4 / 6

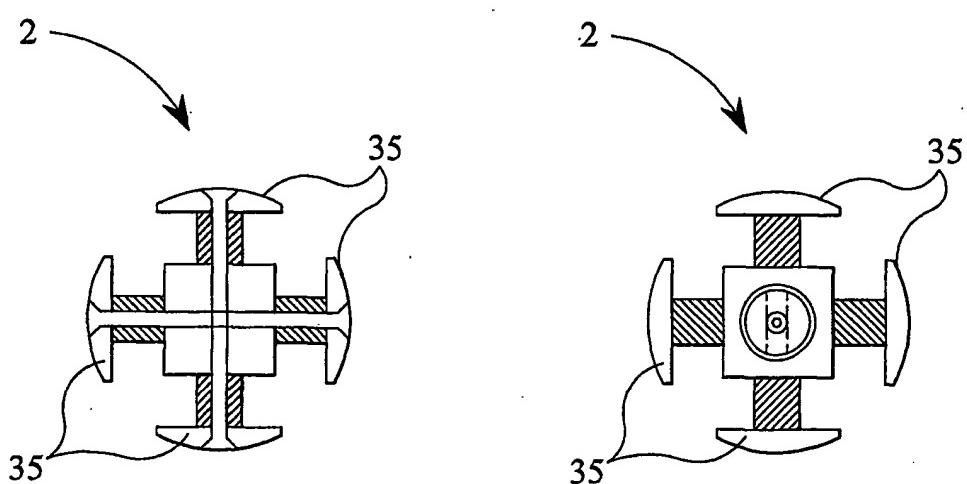
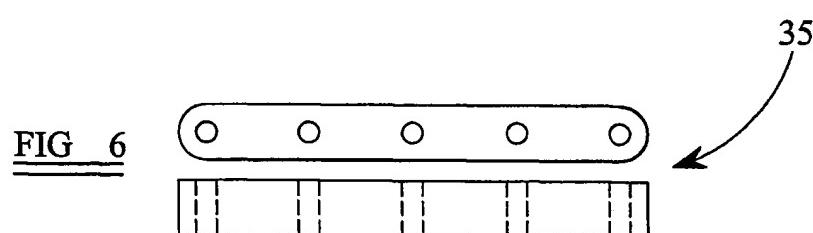
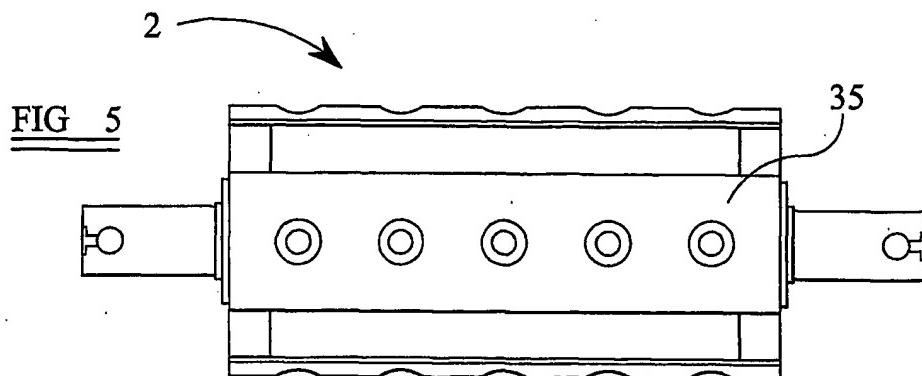
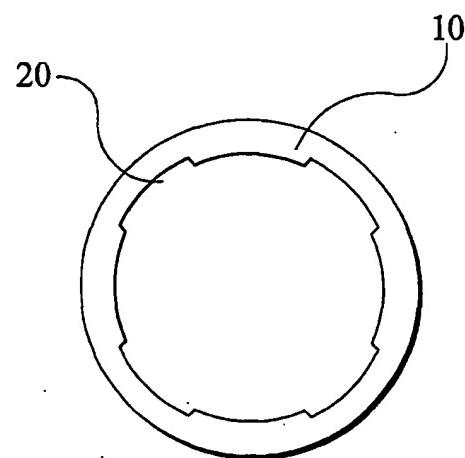
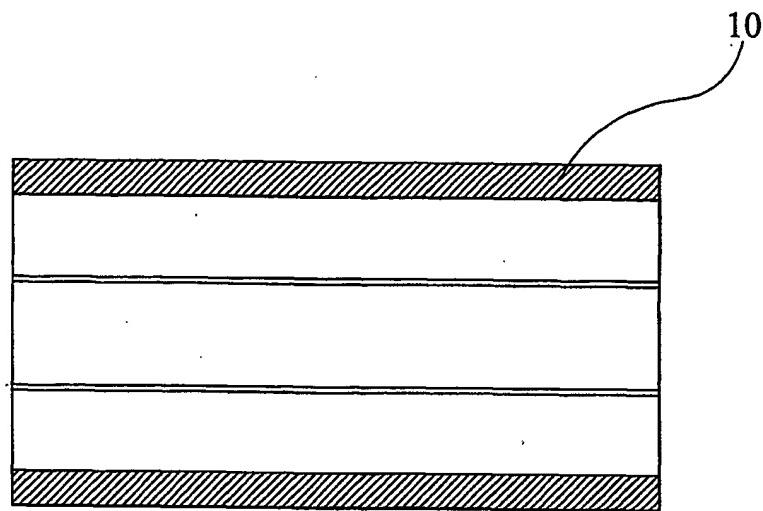
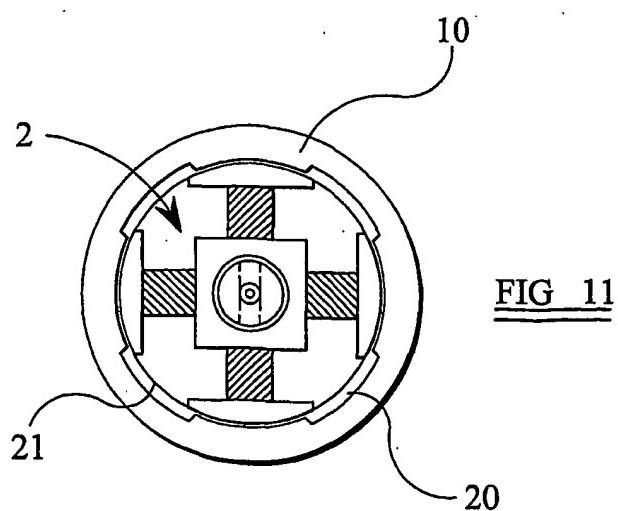


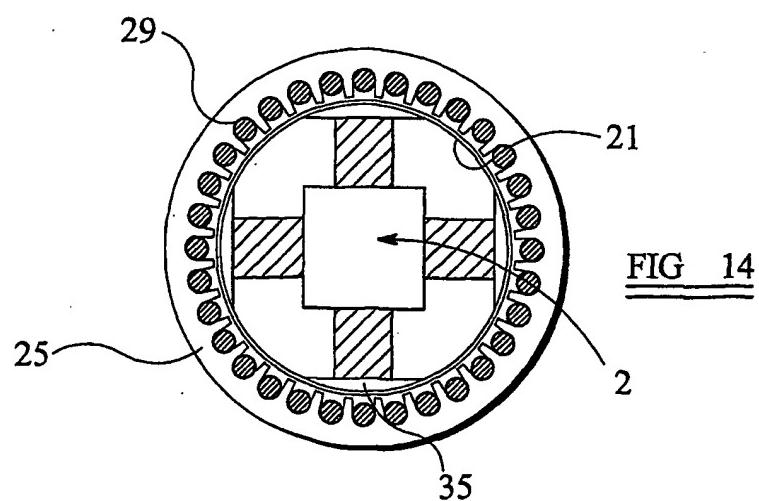
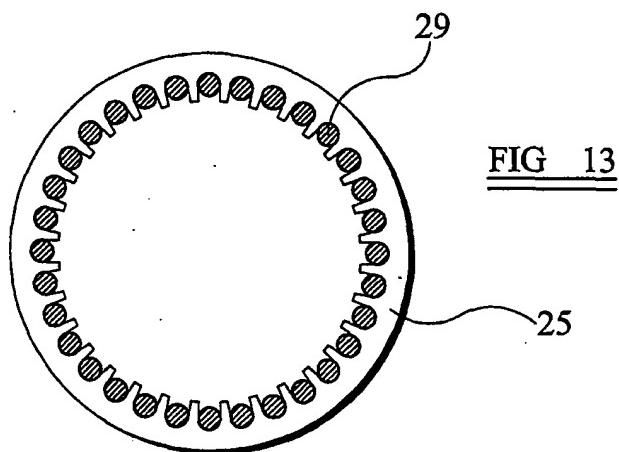
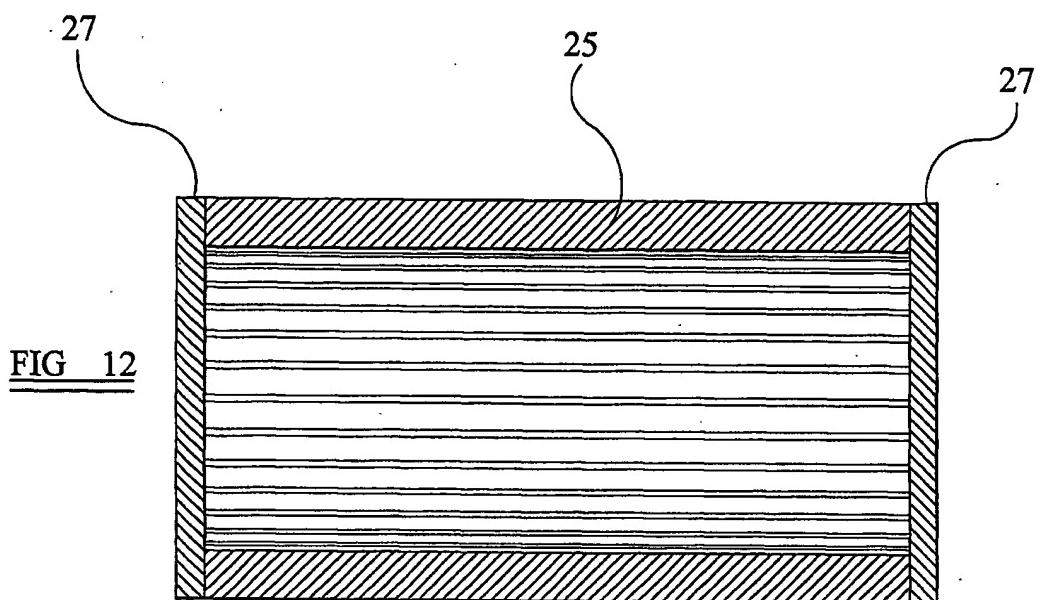
FIG 7

FIG 8

5 / 6

FIG 9FIG 10FIG 11

6 / 6



INTERNATIONAL SEARCH REPORT

PCT/GB 02/01993

A. CLASSIFICATION OF SUBJECT MATTER					
IPC 7	H02K49/04	H02K16/02	H02K7/18	H02K17/34	E21B41/00
	E21B7/06	E21B17/02	F04D29/66	F04D13/10	

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
IPC 7 H02K E21B F04D

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal, WPI Data, PAJ

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	EP 0 607 429 A (SHIKOKU SOGO KENKYUSHO KK ; SATO KAZUNOBU (JP)) 27 July 1994 (1994-07-27) the whole document ---	1-23
X	DE 198 56 224 A (ZAHNRADFABRIK FRIEDRICHSHAFEN) 8 June 2000 (2000-06-08) the whole document ---	1-23
A	EP 0 520 733 A (CAMCO DRILLING GROUP LTD) 30 December 1992 (1992-12-30) column 9, line 56 -column 10, line 12 column 11, line 7 -column 12, line 33 figure 3 --- -/-	1-23

 Further documents are listed in the continuation of box C. Patent family members are listed in annex.

* Special categories of cited documents :

- "A" document defining the general state of the art which is not considered to be of particular relevance
- "E" earlier document but published on or after the international filing date
- "L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)
- "O" document referring to an oral disclosure, use, exhibition or other means
- "P" document published prior to the international filing date but later than the priority date claimed

- "T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
- "X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone
- "Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art
- "&" document member of the same patent family

Date of the actual completion of the international search

Date of mailing of the international search report

16 August 2002

23/08/2002

Name and mailing address of the ISA

European Patent Office, P.B. 5818 Patentlaan 2
NL - 2280 HV Rijswijk
Tel. (+31-70) 340-2040, Tx. 31 651 epo nl,
Fax: (+31-70) 340-3016

Authorized officer

Ramos, H

INTERNATIONAL SEARCH REPORT

PCT/GB 02/01993

C.(Continuation) DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	EP 0 728 908 A (CAMCO DRILLING GROUP LTD) 28 August 1996 (1996-08-28) column 8, line 48 -column 11, line 33 column 12, line 53 -column 13, line 41 figures 4,5 -----	1-23

INTERNATIONAL SEARCH REPORT

PCT/GB 02/01993

Patent document cited in search report		Publication date		Patent family member(s)		Publication date
EP 0607429	A	27-07-1994		JP 2604038 B2 JP 3150054 A DE 69029818 D1 DE 69029818 T2 EP 0607429 A1 WO 9107004 A1		23-04-1997 26-06-1991 06-03-1997 21-08-1997 27-07-1994 16-05-1991
DE 19856224	A	08-06-2000		DE 19856224 A1 WO 0035067 A1 EP 1142086 A1		08-06-2000 15-06-2000 10-10-2001
EP 0520733	A	30-12-1992		AU 666850 B2 AU 1851192 A AU 689696 B2 AU 4575896 A CA 2072228 A1 DE 69211229 D1 DE 69211229 T2 DE 69229963 D1 DE 69229963 T2 EP 0520733 A1 EP 0677640 A1 GB 2257182 A ,B GB 2285651 A ,B NO 922473 A NO 982258 A US 5265682 A AU 647957 B2 AU 2136992 A CA 2077177 A1 DE 69219250 D1 DE 69219250 T2 EP 0530045 A1 GB 2259316 A ,B NO 923369 A US 5553678 A		29-02-1996 07-01-1993 02-04-1998 06-06-1996 26-12-1992 11-07-1996 06-02-1997 14-10-1999 20-04-2000 30-12-1992 18-10-1995 06-01-1993 19-07-1995 28-12-1992 18-05-1998 30-11-1993 31-03-1994 04-03-1993 01-03-1993 28-05-1997 18-12-1997 03-03-1993 10-03-1993 01-03-1993 10-09-1996
EP 0728908	A	28-08-1996		AU 713495 B2 AU 4550496 A CA 2170183 A1 DE 69609744 D1 DE 69609744 T2 EP 0728908 A2 GB 2298217 A ,B NO 960593 A US 5695015 A		02-12-1999 05-09-1996 26-08-1996 21-09-2000 12-04-2001 28-08-1996 28-08-1996 26-08-1996 09-12-1997

**This Page is Inserted by IFW Indexing and Scanning
Operations and is not part of the Official Record**

BEST AVAILABLE IMAGES

Defective images within this document are accurate representations of the original documents submitted by the applicant.

Defects in the images include but are not limited to the items checked:

- BLACK BORDERS**
- IMAGE CUT OFF AT TOP, BOTTOM OR SIDES**
- FADED TEXT OR DRAWING**
- BLURRED OR ILLEGIBLE TEXT OR DRAWING**
- SKEWED/SLANTED IMAGES**
- COLOR OR BLACK AND WHITE PHOTOGRAPHS**
- GRAY SCALE DOCUMENTS**
- LINES OR MARKS ON ORIGINAL DOCUMENT**
- REFERENCE(S) OR EXHIBIT(S) SUBMITTED ARE POOR QUALITY**
- OTHER:** _____

IMAGES ARE BEST AVAILABLE COPY.

As rescanning these documents will not correct the image problems checked, please do not report these problems to the IFW Image Problem Mailbox.